



Numerical Analysis of Micro Pin Fin Heat Sink Using Nano Fluids as Coolants

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Abstract: In this paper a fluid flow and thermal characteristics of micro pin fin heat sink is investigated with the computational fluid dynamics, by this with three fins geometries (square, triangular and circular) and Nano fluids is used as a cooling fluid, since the flow and heat transfer have been studied with two types of Nano fluids (Diamond-water and Alumina-water) in addition to the base fluid water. The results obtained indicated that, using of Nano fluids instead of water as a coolant leads to enhanced heat transfer performance Also performance of heat sink is good. Here for the analysis of heat sink, the model has been designed in the CATIA V5 software, and for the fluid flow and numerical analysis ANSYS FLUENT 16.2 software is used.

Keywords: Heat sink, Nano fluids, 3D Fins Geometry, Reynolds number.

I. INTRODUCTION

Increasing in the electronic devices in the modern days has the wide range of usages over the other devices with minimizing of time and also increasing the efficiency of work. The microelectronic devices like CPU's, LED's and many more dissipates heat from the chips hence inside the devices more heat generation has disadvantages of the devices for that its necessitate to cool the devices by minimizing the heat generated, and increasing the heat removal rate from the chips is only possible by the cooling. For cooling of the electronic devices keeps the components of devices stable and effective. Here liquid cooling is more effective than the air cooling and also the fluids will carry large amount of heat transfer rate. Looking at the background study of the liquid cooling of the microelectronic devices its came to know that the using of Nano fluids instead of base fluid water has the more advantage and also Nano fluids can carry large amount of heat by this usage of Nano fluids in the heat sink is more benefits. Nano fluids is a kind of liquids containing small quantity of Nano sized particles (usually less than 100nm) Micro channel heat sinks due to their small mass and volume as well as larger area to volume ratio are very attractive for cooling of high heat flux chips. To improve the cooling performance and temperature uniformity of

these devices, many new innovative ideas have been proposed such as using pin fins with different shapes and different arrangements. Fins as an extended surfaces play an important role in enhancing heat transfer process by increasing the area of heat transfer. There are many researches in literature made to study the micro pin heat sink with different fins geometries and to study the heat transfer rate in Nano fluids. Mustaq Ismael Hasan [1] has studied a micro pin fin heat sink is numerically investigated with three fins geometries (square, triangular and circular) in addition to the un finned micro channel heat sink. Nano fluid is used as a cooling fluid, since the flow and heat transfer have been studied with two types of Nano fluids (Diamond-water and Al_2O_3 -water) in addition to the pure water. S.Subramanian, K.S.Shridhar, C.K.Umesh [2] researched this work is about the investigation for the performance enhancement by modifying the oblique fin geometry. Seven variants of micro channel geometries have been explored using three dimensional numerical simulations. The variants are plate fin, inline pin fin, staggered pin fin, oblique fin. Piyush Laad, Bhushan Akhare, Praveen Chaurasia[3] studied about The effect of the pin-fin shapes on the overall performance of the heat sink with inline and staggered arrangement is studied in this paper. Six different shapes of fins rectangle, trapezoidal, rectangular interrupted, square, circular inline and staggered are

subjected to study in this paper. Ravishankar Prasad Maurya, Ankesh Kumar Pataskar[4] has researched investigation of performance of heat sink with variable pin attached along with uniform fin. The present work deals with the computational analysis of flow (CFD analysis) and heat transfer with pin fin. In the present work a diamond shaped pin fin are used and is investigated under constant heat flux condition at the place of plane fin or elliptical pin fin. Asif Afzal*, A.D. Mohammed Samee, R.K. Abdul Razak, M.K. Ramis [5] has worked on experimental comparative thermal analysis of Multi Walled Carbon Nano Tubes (MWCNT) in rectangular mini channels. 0.01% volume concentration of MWCNT nanoparticles were suspended in water as base fluid to obtain MWCNT-water Nano fluid. Venkatesh Saravanan, Chitradurga K Umesh, Doddamani Hithaish, Kankanahalli Seetharam[6] they investigated comparison of fluid flow and heat transfer characteristics for micro pin fin heat sink and micro channel pin fin heat sink with un finned micro channel heat sink. A Three Dimensional heat sink with water as coolant subjected to constant heat flux $10\text{W}/\text{cm}^2$, for Reynolds number ranging between 100-900 is considered for study.

After reviewing of all literature surveys its came to know that how to design a model of heat sink should be and how to cool the micro pin fin heat sink by using Nano fluids comparing it to the other fluids. After all surveys its concluded that for effective transfer of heat in the electronic systems it should be based on the how the geometry of the heat sink design made and how much capacity of heat it should withstand. By simulating the designed model in in the ANSYS FLUENT software the variation of velocity, temperature and pressure inside the heat sink pin fin has been estimated and compared. Most of the studies in literature made to investigate heat transfer and flow

characteristics in pin fin heat sink with pure fluids and as the author knowledge there are no researches available in which the Nano fluid used as a coolant in mini and micro pin fin heat sink. In this paper two types of Nano fluids were used as a coolant in micro pin fin heat sink and the effect of using these Nano fluids instead of pure water on the thermal and hydrodynamic performance of pin fin heat sink is numerically investigated.

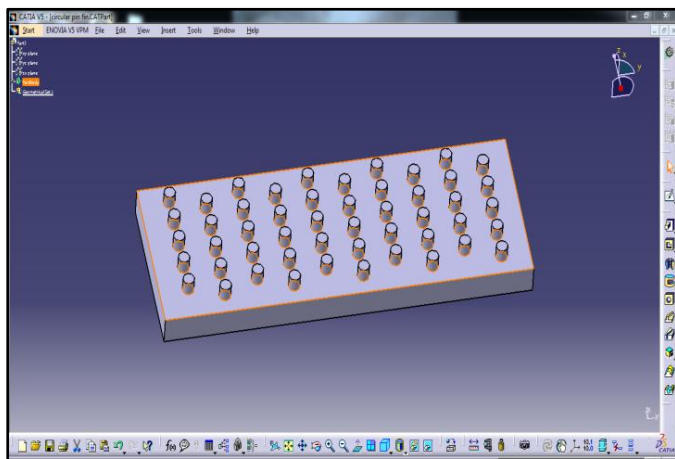
II. PROBLEM FORMULATION

Micro pin fin heat sink was designed with different fin geometries and also fins dimension was varied. Velocity of fluid flow is determined by using Reynolds number laminar flow and the heat transfer rate of micro pin heat sink is calculated using CFD.

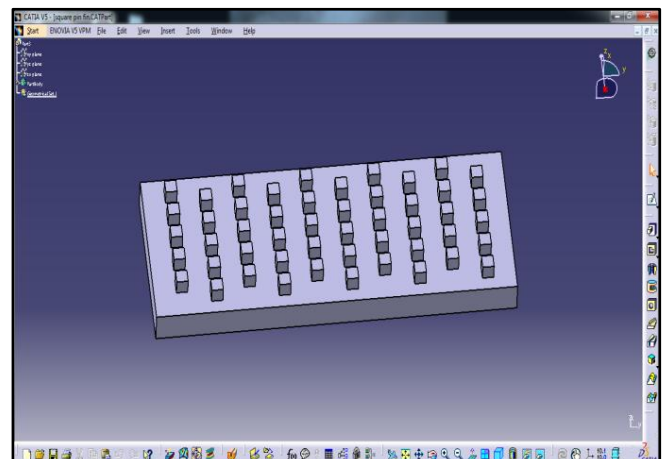
Also by using different Nano fluids comparing with base fluid as water showing which Nano fluids makes effective cooling of micro pin fin heat sink in the project by ANSYS FLUENT software.

III. PHYSICAL GEOMETRY

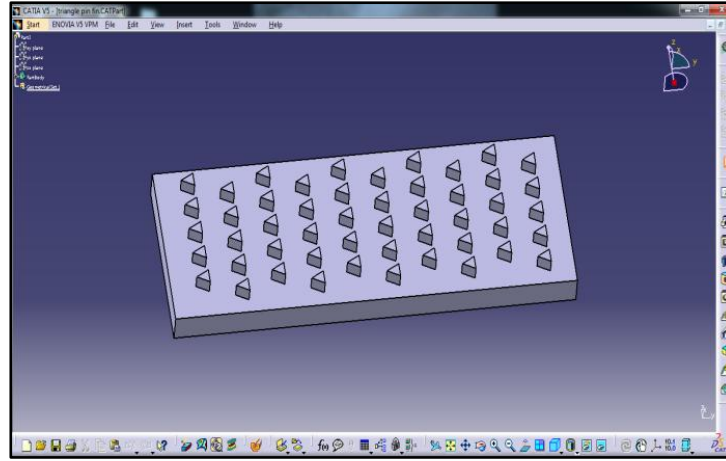
The model studied in this paper is 3D micro pin fin heat sink consists of 50 fins with three shapes (square, circular and triangular) fins with staggered arrangement. The length of heat sink is 16 mm and its width and height are 6 mm and 1 mm respectively. All fins have same height 0.5 mm and base dimensions width of fin = 0.5 mm, Depth of fin = 0.5 mm. The physical model represents the studied micro pin fin heat sink with three geometries of fins. The fins are arranged in staggered configuration which is better than inline configuration from the heat transfer point of view as it used by many references the values of fins spacing is 0.5 mm. this geometry is designed in CATIA V5 R20 software.



(a)



(b)



(c)

Figure 1: 3D Physical Geometries of (a) Circular pin fin heat sink (b) Square pin fin heat sink (c) Triangular pin fin heat sink.

The base material of heat sink used is Aluminium and the Nano fluids introduced inside the heat sink are Diamond-water and Alumina-water comparing with the water fluid. Velocity of flow is taken 0.069m/s based on Reynolds number at the temperature of 20°C at the inlet of the heat sink. Constant heat flux of 1W is applied on the bottom face, all the fins undergone to convection.

IV. GOVERNING EQUATIONS

For modeling, Nano fluid is treated as a single phase fluid. This assumption can be applicable since the particles are ultra-fine and they are easily fluidized. Moreover, the particle volume fraction in Nano fluid is usually low. Under such assumptions the governing equations for the Nano fluid flow and heat transfer are greatly simplified and local fluid and particles are in thermal equilibrium. For steady state, 3-D, incompressible and laminar flow the following equations are solved to calculate the distributions of velocity and temperature.

Continuity equation:

$$\nabla V=0 \quad (1)$$

Momentum equation:

$$\rho(V\nabla V)=-\nabla P + \nabla(\mu\nabla V) \quad (2)$$

Energy equation:

$$\rho C_p(V\nabla T) = k\nabla^2 T \quad (3)$$

4.1 Boundary Conditions

- At the inlet: velocity inlet taken ranges from 0.069 m/s at ambient temperature 20°C.

- At the outlet: pressure outlet is taken (flow assumed to be fully developed).

- At the lower surfaces of the heat sink constant wall temperature is applied ($T=100^\circ\text{C}$).

- For the Bottom side of the heat sink: heat flux = 1W.

- No-slip boundary condition is applied to all fluid-solid interfaces.

Outer surface of the heat sink model (right, left, and top) are assumed to be adiabatic walls.

The hydraulic diameter is calculated by using Equation (4).

$$D_h = \frac{4ab}{2(a+b)} \quad (4)$$

Reynolds number is calculated using Equation (5).

$$R_e = \frac{\rho D_h v}{\mu} \quad (5)$$

Heat transfer co-efficient is calculated using the relation (6).

$$Q = hA\Delta T \quad (6)$$

V. RESULTS AND DISCUSSION

5.1 Validation:

The results obtained from the present work is validated with the investigated Model [1] of the paper, and the below Fig 2 validates the results hence the present thesis are in good agreement.

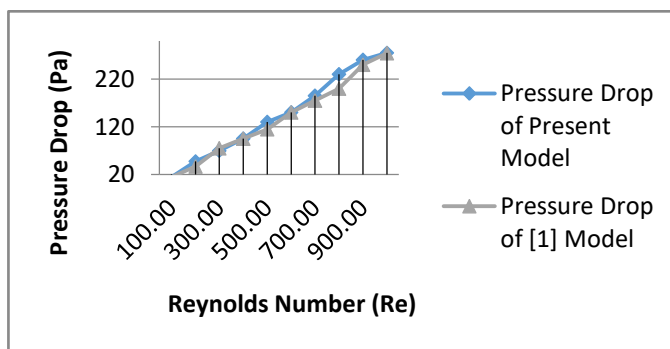
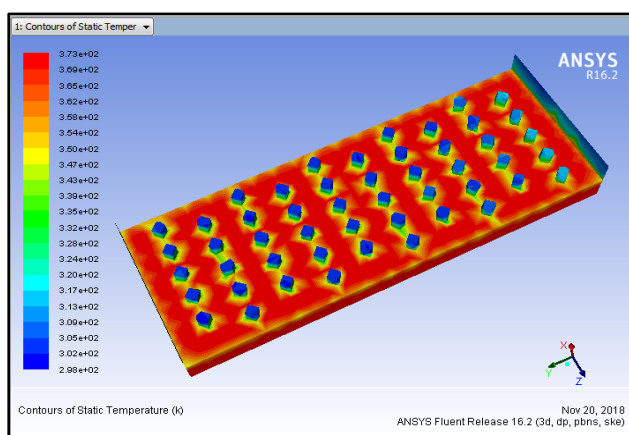
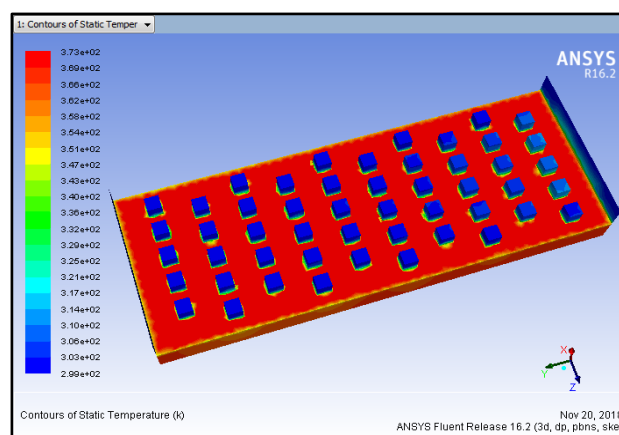


Figure 2: Variation of Pressure Drop with the Different Reynolds Number.

5.2 Numerical results:



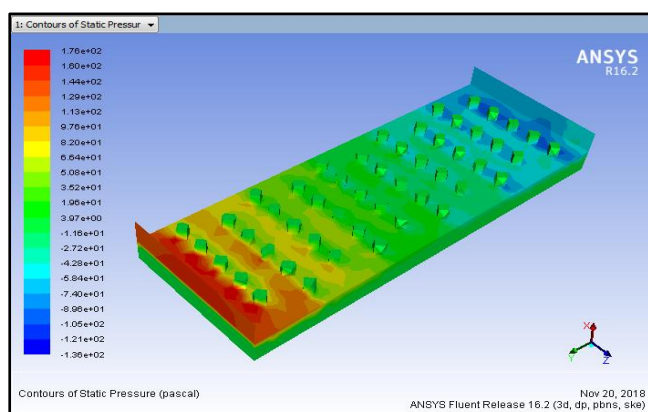
(a)



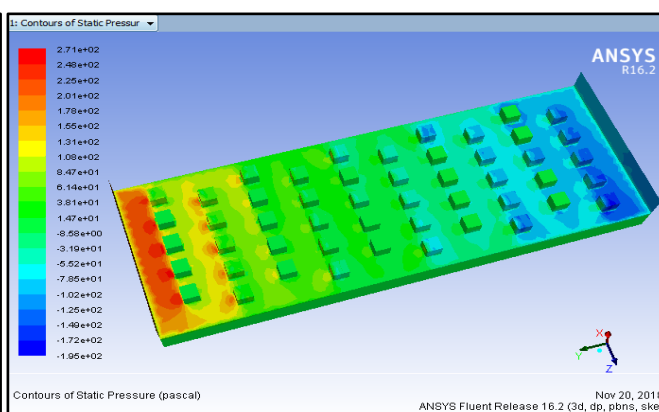
(b)

Figure 3: Temperature Contours of (a) Circular and (b) Square Geometry.

In the above figures Fig-3 (a) and (b) shows that the temperature distribution at the Reynolds number $Re=1000$, in the laminar flow region.



(c)



(d)

Figure 3: Pressure Contours of (c) Circular and (d) Square Geometry.

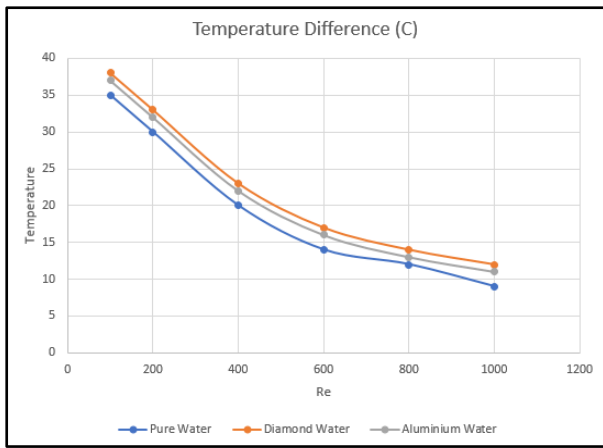


Figure 4: Temperature difference v/s Re in circular pin fins

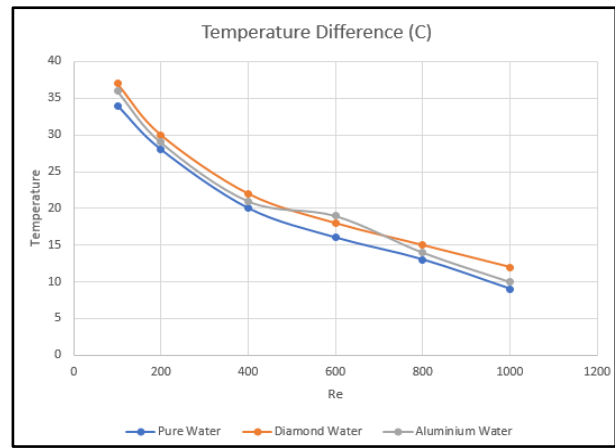


Figure 5: Temperature difference v/s Re in square pin fins

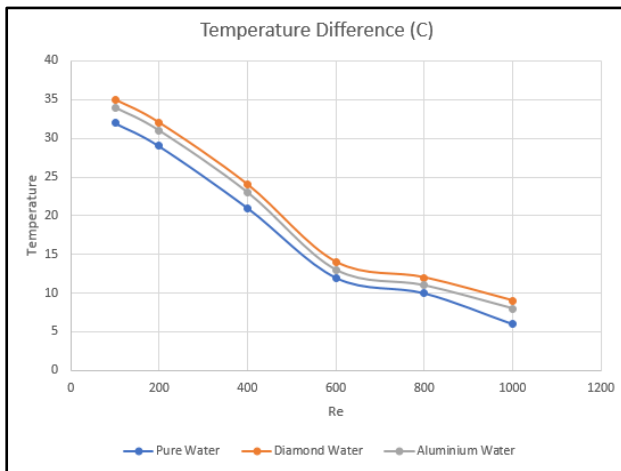


Figure 6: Temperature difference v/s Re in triangular pin fins.

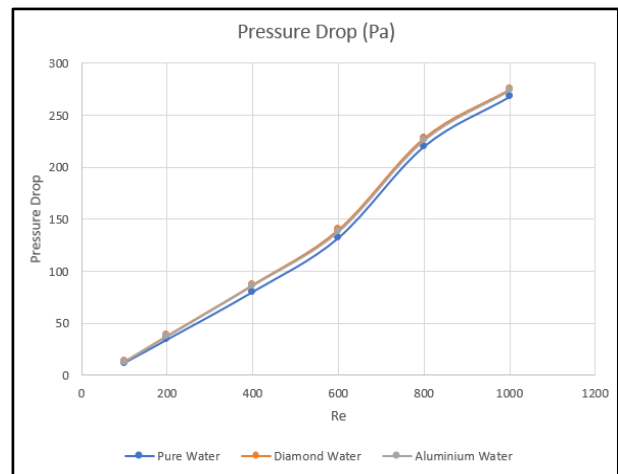


Figure 7: Pressure drop v/s Re in circular pin fins.

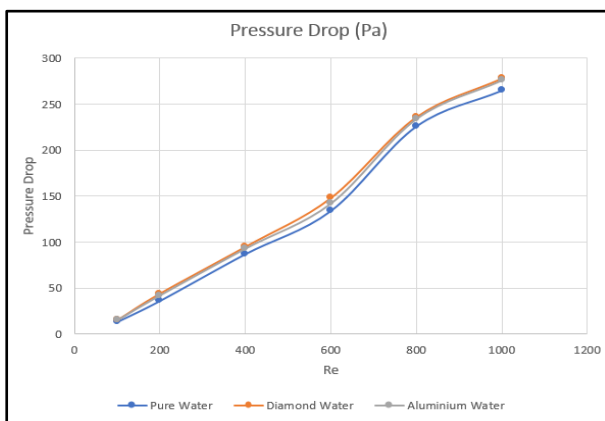


Figure 8: Pressure drop v/s Re in square pin fins.

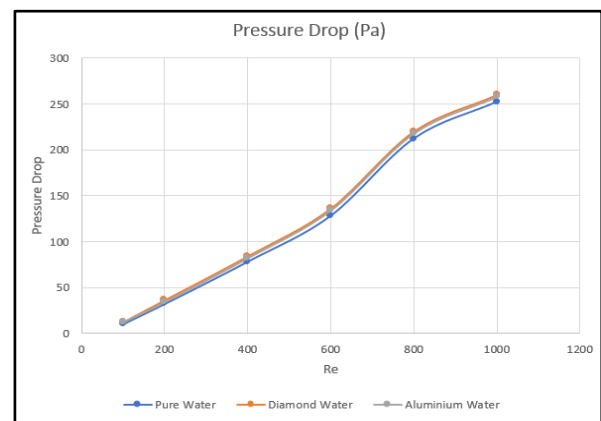


Figure 9: Pressure drop v/s Re in triangular pin fins.

From the Fig-3 (c) and (d) which shows the pressure distributions in the micro pin fin heat sink i.e. the static pressure contours occurs in the circular and square fin pin geometry. And the Figures 4, 5 and 6, which explains about the temperature difference inside the micro pin fin heat sink varied with the Reynolds number using different Nano fluids like Diamond-water, and Alumina-water compared with the water

fluid in the circular, square and triangular geometries. Here the circular geometry has the more temperature difference also diamond water has the more heat carrying capacity than the other fluids.

From the Figures 7, 8, and 9, it is observed that the pressure drop inside the square pin fin heat sink geometry has found to be more compared to the other

geometries hence diamond-water Nano fluid which can be effectively cools the heat sink.

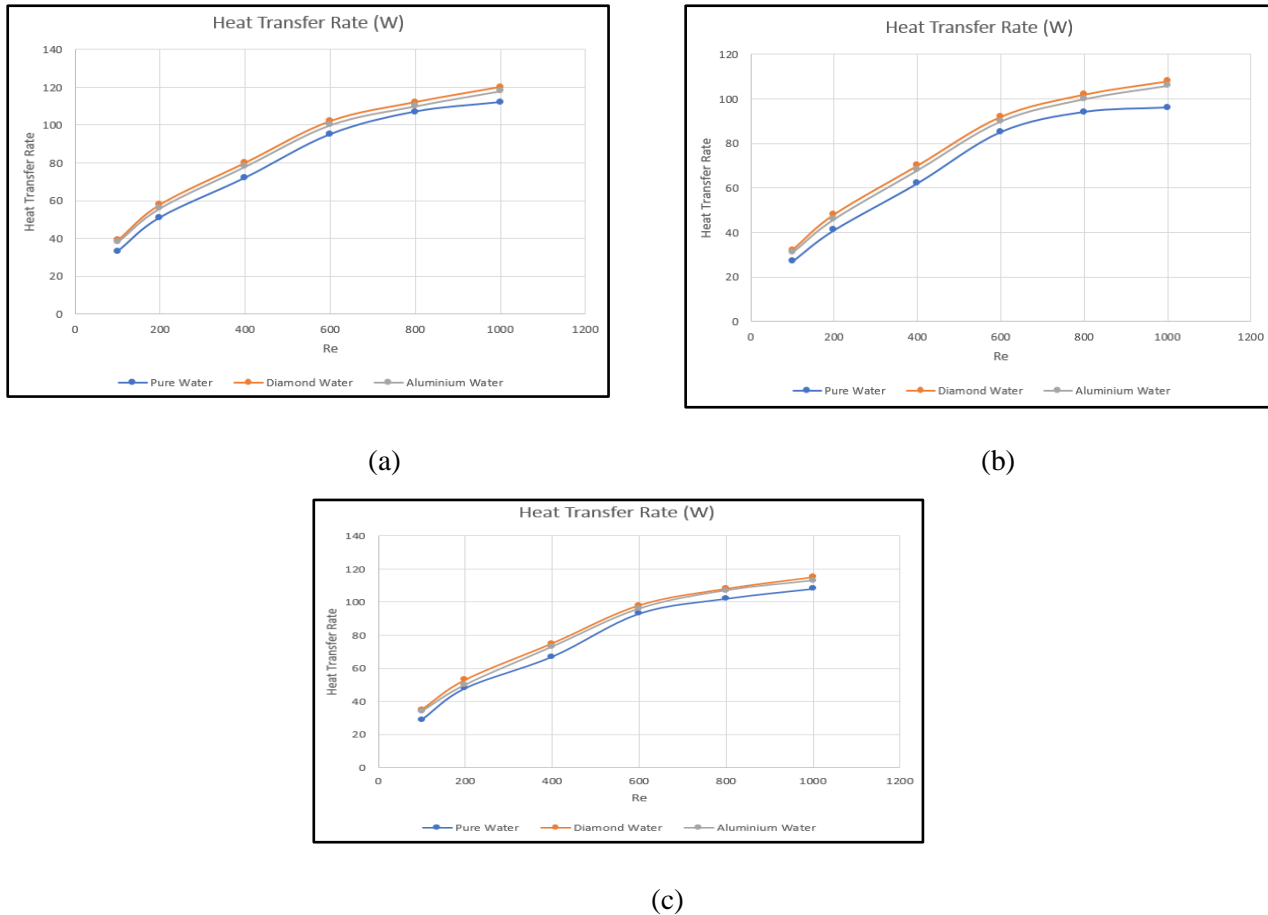


Figure 10: Heat transfer rate v/s Re in (a) Circular pins (b) Square pin fins (c) Triangle pins

From the Figure-10 (a), (b) and (c) it is observed that the heat transfer rate from the micro pin fin heat sink using Nano fluids with different fin geometry shapes, the circular pin fins geometry has the more heat removal rate from the heat sink than the other geometries also Nano fluids used i.e. Diamond-water and Alumina-water has the effective cooling characteristics over the base fluid water.

VI. CONCLUSION

Three types of fluids are used by varying the Reynolds number in different fin geometries like circular, square and triangular.

- In the numerical analysis of Micro pin fin heat sink using nano fluids has the greater range of heat removal and also enhanced performance of heat sink
- Nano fluids are better than any other fluids from heat transfer point of view these can carry larger amount of heat.
- By increasing the volumetric concentration of Nano fluids both heat transfer rate and pressure drop have been increased dramatically.
- At constant temperature as a thermal boundary condition and for all values of Re and for both Nano

fluids studied, the circular fins give higher heat transfer rate compared with other fins. And the square fins caused higher pressure drop.

In future scope of the analysis of heat sink, removing of heat generated can be increased by increasing the fins geometry i.e. varying the fins length, width and depth.

Nomenclature

- a- Length of heat sink, mm.
- Re- Reynolds number.
- b-Width of heat sink, mm.
- CPU-Control Processing Unit.
- C_p -Specific heat $Jkg^{-1}K^{-1}$.
- LED-Light Emitting Diode.
- K - Thermal Conductivity $Wm^{-1}K^{-1}$.
- 2D- Two Dimensional.
- q - Heat flux input, Wm^{-2} .
- 3D- Three Dimensional.
- Q -Total heat Transfer, W.
- Al_2O_3 -Aluminium Oxide.
- P- Pressure, Pa.

Greek symbols

- v -Velocity of liquid flow, ms^{-1} .
- Δ - Difference in value.

Dh- Hydraulic diameter mm.
 ρ - Density, Kg m^{-3} .
 T -Temperature $^{\circ}\text{C}$.

Conflicts of Interests

The author does not have any conflict of interest regarding the publication of paper

Acknowledgement

The author expresses his earnest thanks to the reviewers of this paper for their valuable remarks and suggestions for improvement of paper.

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